LAB 1. A 2D quarter model of a plate with notches. Stress concentration factor as a function of the notch radius.

- 1. Geometry of a quarter model of a thin plate with the notch r_1 (Fig. 1):
- a) create a rectangle (X- coordinates: 0 to $a+r_1$, Y- coordinates: 0 to $3a+r_1$, Fig. 2)
- b) display and move the working plane (X, Y, Z Offsets) : $a+r_1$, 0, 0 (Fig. 3)
- c) create a circle (outer radius: r_1) (Fig. 4)





d) subtract the circle from the rectangle (Fig. 5): (*pick or enter base areas from which to subtract ->* select the rectangle (-> ok), *pick or enter areas to be subtracted* -> select the circle (-> ok)

2. Choose a finite element: PLANE182 (Quad 4 node) with the plane stress option and the enhanced strain technology (Fig. 6) :

3. Define linear isotropic material properties: E (Young's modulus) and ν (Poisson's ratio) (Fig. 7)

4. Save the database (Utility Menu>File>Save As..., *.db)

5. Define the discretisation density: calculate the element size on keypoints using the parameter m (from **Tables 1** and **2**, initial value of m = 1) and mesh the plate (Fig. 8)

6. Save the mesh as an image (Plot> Elements, PlotCtrls> Redirect Plots-> To JPEG File ...)

7. Save in Table 1:

numbers of finite elements (NOE) and nodes (NON) (Utility Menu>List>

Status> Global Status)

- the distance *d* between the two nodes close to the notch tip (*Utility Menu>List>Picked Entities+*) (Fig. 9)

- 8. Apply boundary conditions: symmetry conditions (Fig. 10) and pressure of 10 MPa (Fig. 11)
- 9. Solve (Solution-> Solve-> Current LS)



10. Plot and save as an image a contour map of the displacement magnitude (Main Menu>General Postproc> Read Results> First Set, *Main Menu> General Postproc> Plot Results> Contour Plot> Nodal Solu> DOF Solution> Displacement Vector sum 'USUM'*) and write the maximum value of USUM in **Table 1.**

11. Plot and save as images contour maps of stress components SX, SY, SXY and Von Mises stress SEQV (Main Menu>General Postproc>Plot Results>Contour Plot>Nodal Solu>Stress ...)

12. Find stress components SX, SY, SXY at the node located at the notch tip (point B, Fig. 8) (*Main Menu>General Postproc> Query Result> Subgrid Solu*) and save the data in **Table 1**:

13. Decrease the value of m and repeat steps 5, 6, 7, 9 and 12.

14. Change a finite element to PLANE 183 (the 'Quad 8 node' element with mid-side nodes is a higher order version of PLANE 182)

15. Repeat steps 5, 7, 9 and 12 starting from m = 1 and fill the **Table 2**.

16. Find and underline in the worksheet the optimal value of 'm' separately for both element types. Consider the following criteria:

- normal stress (SX) at the notch tip near to zero,

- shear stress (SXY) at the notch tip near to zero,

- the number of nodes as small as possible.

(Final report should also include three graphs of SX(d), SY(d), SXY(d) for both element types, where d is the distance shown in Fig. 9)

17. Perform FE analyses of plates with notches r_1 , r_2 , and r_3 . Choose one type of a finite element (4-node or 8-node) and the relevant value of m determined at point 16. For each analysis write in the worksheet or save on the disc:



- image of finite elements,

- contour map of the displacement magnitude (USUM),

- contour map of the equivalent Von Mises stress (SEQV),

- graph with distribution of the stress components between points A and B (SX, SY, SXY, SEQV). To plot the graph: · select points A and B (Main Menu>General Postproc>Path Operations>Define Path>By Nodes (ok),

Name= path1, nSets =30, nDiv=200)

 \cdot select stress components in sequence (Main Menu>General Postproc>Path Operations>Map onto Path (ok) – the field 'Lab' can be left blank)

 \cdot select stress components to plot (Main Menu>General Postproc>Path Operations>Plot Path Item>On Graph) - calculation of a stress concentration factor α :

$$\alpha = \frac{SY_B}{p \cdot \left(\frac{a + r_1}{a}\right)}$$

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(r₁ for all radii (see Fig. 12))

18. Plot $\alpha(r)$ as a graph (Excel).

19. Perform analyses of a plate with the notch r_1 for the plane strain and axis-symmetry conditions (Fig. 13). Use the same type of a finite element and *m* value chosen at point 17. Calculate α values for 3 different element options (to calculate α for the axis-symmetry case divide the stress by $p \cdot ((a+r_1)/a)^2$ instead of $p \cdot (a+r_1)/a$).



